Hochschule Düsseldorf University of Applied Sciences Zentrum für Innovative Energiesysteme Centre of Innovative Energy Systems



Efficiency studies on a heat pump system with a stratified storage tank for space heating and domestic hot water based on hardware in the loop tests

1

Maximilian Kampmann, M.Sc. Scientific Assistant

## Introduction

#### Background of the work

Fraunhofer study from 2011 (Miara et al. 2011): Stratified storage systems in building heat supply (for about 200 m<sup>2</sup> living space) comparatively inefficient - reasons:

- Incorrect positioning of temperature sensors
- Inaccurate control parameters
- Inadequate loading strategy

Annual Performance Factors Fraunhofer 2011



## Introduction

#### Aim of the investigation

- Elaboration of an efficient loading management of stratified storage systems
- Investigation of efficiency parameters
- Investigation of the heat pump cycle rate (compressor lifetime and grid flexibility)
- Elaboration of interactions of the variables

#### HiL-experiments are carried out with the heat pump

Influences of the transient behavior on the stratified storage loading

## Heat Supply System



Geothermal probe

## Heat Supply System



Geothermal probe

(Nominal supply temperature of space heating: 35 °C)

## Heat Supply System

#### Heat sinks (IEA Annex 38 Task 44)

Building (SFH 45):

- Single family house with 140 m<sup>2</sup>
- Heat demand: 6500 kWh/a
- Renovated building with good thermal quality (Dott et al. 2013)

#### Domestic hot water:

- 2130 kWh/a or 140 liters of DHW per day
- Corresponds to a household with approx. 3 to 4 persons

#### Weather data (IEA Annex 38 Task 44)

Central European climate (Strasbourg weather data)

#### Heat source

Geothermal probe dimensioned according to VDI guideline 4640

#### Storage

Sailer stratified tank, parameter estimation according to DIN EN 12977-3

#### Heat pump

Viessmann Vitocal 300-G (Year of manufacture 2014) Nominal power about 5.5 kW

#### Models from CARNOT-toolbox 7.0 (© Solar-Institut Jülich) in Simulink

## **Experimental design**

# Tests on a hardware in the loop test bench

- Heat pump as real component
- Rest of the system is modeled (Simulink - Carnot Toolbox)



#### Test scope: 24-hours

Constant boundary conditions for each test – typical day for the heating period

Typical day derived from an annual simulation of the system

Outside temperature:	9 °C
Space heat demand:	35 kWh (86 %)
Domestic hot water:	6 kWh (14 %)

Targets

**Q**<sub>St,loss</sub> Heat pump 52 °C 47 °C COP cycle rate 45 °C 30 °C Space heating

- COP of the Heat pump
- Storage heat loss
- Heat pump cycle rate

## Influencing variables

#### Parameters

- Relative sensor heights for SH and DHW zone
- Set temperature of the SH zone
- Flow rate in the secondary circuit (heating circuit) of the HP
- Storage volume

#### Limits

Symbol	Limits		Unit
	Min	Max	Onit
h <sub>SH</sub>	0,125	0,312	-
h <sub>DHW</sub>	0,625	0,812	-
V <sub>St</sub>	531	731	Ι
T <sub>SH,set</sub>	40	45	°C
V <sub>Sec</sub>	800	1250	l/h



## Design of experiments (DoE)

#### Fractional factorial experimental design (as little experimental effort as possible)

- Targeted omission of corner points to shorten the experimental design (knowledge gain remains almost the same)
- $2^{5-1} = 16$  experimental points

#### **Central point** (measured 5 times in total)

- Control of the regression •
- Scattering of the test points

#### **Regression function**

![](_page_9_Figure_8.jpeg)

$$Target = c_0 + \sum_{i=1}^{n_f} c_i x_i + \sum_{i=1}^{n_f-1} \sum_{j=i+1}^{n_f} c_{ij} x_i x_j + \varepsilon$$
  
Linear effects 2-way interactions

![](_page_10_Figure_1.jpeg)

Bandwidth COP: ± 5%

in absolute values. 3,6 ... 3,95

### Sensor positions:

High positioning decreases mean storage tank temperature

Decreases upper storage tank temperature

 $\rightarrow$  tap temperature remains > 45 °C

![](_page_11_Figure_5.jpeg)

#### Flow rate of the HP (secondary/heating circuit):

High flow rate creates mixing in the storage tank (Glembin et al. 2015)

Higher mean condenser temperatures increase HP temperature lift slightly

![](_page_12_Figure_4.jpeg)

## **Results Cycle rate**

![](_page_13_Figure_1.jpeg)

Cycle rate in contour plot scales with storage volume: ± 100 I correspond to ± 0.15 1/h

## Conclusion

## Application of the determined effects to the annual performance factor

Range of the effects of the variables:  $\pm 0.25$ 

• COP and APF comparable in both investigations:

heating energy of the HP el. energy of compressor, pumps, control

- Time periods different: day vs. year
- Particularly poor settings of the influencing variables not tested here
- Measured baseline value in (Miara et al. 2011) low, due to several systems operating incorrectly

![](_page_14_Figure_8.jpeg)

## Conclusion

#### Set sensor positions high if possible

Positive effects on COP, cycle rate and storage heat loss

#### Temperature of the SH-zone

- High temperature lowers the HP cycle rate
- High temperature has negative impact on COP and storage heat loss
- Hence: use high sensor positions to lower the cycle rate and choose a low storage temperature if possible

#### Storage volume

- Rises the storage heat loss and lowers the HP cycle rate
- Dimension the storage tank to the required heat demand

#### Small flow rate in the HP secondary circuit

### Sources

M. Miara, D. Günther, T. Kramer, T. Oltersdorf und J. Wapler, "WP Effizienz - Messtechnische Untersuchung von Wärmepumpenanlagen zur Analyse und Bewertung der Effizienz im realen Betrieb," Fraunhofer-Institut für Solare Energiesysteme ISE, Freiburg, 2011.

J. Glembin, C. Büttner, J. Steinweg und G. Rockendorf, "Thermal Storage Tanks in High Efficiency Heat Pump Systems – Optimized Installation and Operation Parameters," *Energy Procedia*, Jg. 73, S. 331–340, 2015, doi: 10.1016/j.egypro.2015.07.700.

J. Glembin, C. Büttner, J. Steinweg und G. Rockendorf, "Optimal Connection of Heat Pump and Solar Buffer Storage under Different Boundary Conditions," *Energy Procedia*, Jg. 91, S. 145–154, 2016, doi: 10.1016/j.egypro.2016.06.190.

M. Y. Haller, R. Haberl, I. Mojic und E. Frank, "Hydraulic Integration and Control of Heat Pump and Combi-storage: Same Components, Big Differences," *Energy Procedia*, Jg. 48, S. 571–580, 2014, doi: 10.1016/j.egypro.2014.02.067.

W. El-Baz, P. Tzscheutschler und U. Wagner, "Experimental Study and Modeling of Ground-Source Heat Pumps with Combi-Storage in Buildings," *Energies*, Jg. 11, Nr. 5, S. 1174, 2018, doi: 10.3390/en11051174.

R. Dott, M. Y. Haller, J. Ruschenburg, F. Ochs und J. Bony, "The Reference Framework for System Simulations of the IEA SHC Task 44 / HPP Annex 38: Part B: Buildings and Space Heat Load," Institut Energie am Bau - Fachhochschule Nordwestschweiz, Muttenz, Schweiz, 2013.

Haberl, Robert; Haller, Michel Y.; Papillon, Philippe; Chèze, David; Persson, Tomas; Bales, Chris (2015): Testing of combined heating systems for small houses: Im-proved procedures for whole system test methods. Institut für Solartechnik SPF, Hochschule für Technik HSR. Rapperswil, Schweiz.

## Questions?

![](_page_18_Figure_1.jpeg)

#### Legend:

 $h_{SH}$  - Sensor position SH-Zone  $h_{DHW}$ - Sensor position DHW-Zone  $V_{St}$  - Storage volume  $\dot{V}_{Sec}$  - Flow rate secondary circuit  $T_{SH,set}$  - Set temperature SH-Zone

![](_page_19_Figure_1.jpeg)

#### Legend:

 $h_{SH}$  - Sensor position SH-Zone  $h_{DHW}$ - Sensor position DHW-Zone  $V_{St}$  - Storage volume  $\dot{V}_{Sec}$  - Flow rate secondary circuit  $T_{SH,set}$  - Set temperature SH-Zone

## Results Cycle rate

![](_page_20_Figure_1.jpeg)

## **Results Storage heat loss**

![](_page_21_Figure_1.jpeg)

## **Results Storage heat loss**

![](_page_22_Figure_1.jpeg)

Heat loss in contour plot scaled with storage volume: ± 100 I correspond to ± 7,5 W