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Parallel operation of two heat sources in a DSHP

CARNOT User Meeting 2023

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- 1. Introduction**
- 2. Modelling approach**
- 3. First results**

1. Introduction

Dual-Source Heat Pumps: Concepts

- **To benefit from advantages and limit disadvantages: Heat Pumps using multiple heat sources, e.g.:**
 - Air + solar
 - Ground + solar
 - Air + ground
 - More exotic applications like water + ground, waste heat + air etc.
- **Reduce size from ground source while achieving high efficiencies and limit noise**
- **Different hydraulic approaches: Several heat pump systems, brine cycles for heat sources, etc.**

1. Introduction

Dual-Source Heat Pumps: Aims

- **Reduce size: reduced annual load on ground source by using air source in spring, autumn, summer**
 - Reduces energy extracted from the soil (BHEs, horizontal, etc.)
- **Maintain high efficiency: switch to beneficial heat source according to ambient temperatures**
 - e.g. air source during high ambient temperatures, ground source during cold winter days
- **Reduce noise: switching to ground source during sensible times (e.g. night operation)**
 - reduces ventilator noise but not necessarily compressor noise (depends on the system)

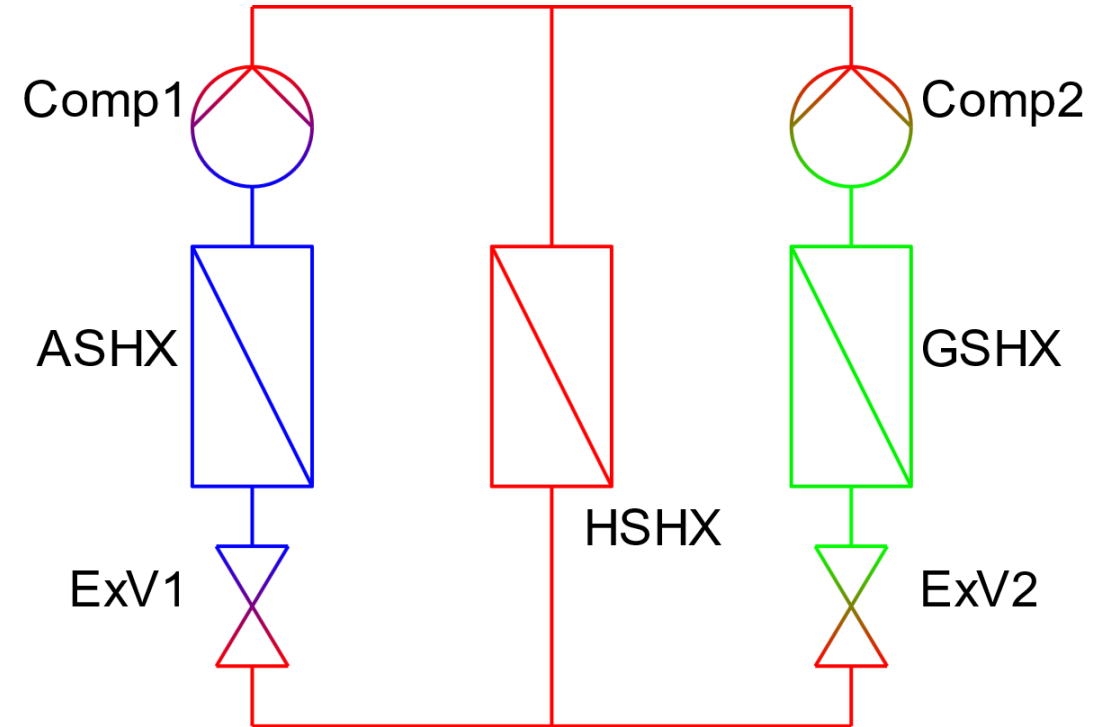
→ **Can in parts contradict themselves: optimization according to purpose necessary**

1. Introduction

Novel hybrid heat pump system



- **Is it possible to operate both heat sources in parallel?**
- Can reduce power load on each heat source
- Non-trivial for the refrigerant cycle: for full flexibility and efficiency, separate evaporation pressures are needed



— Low pressure — Medium pressure — High pressure

2. Modelling approach

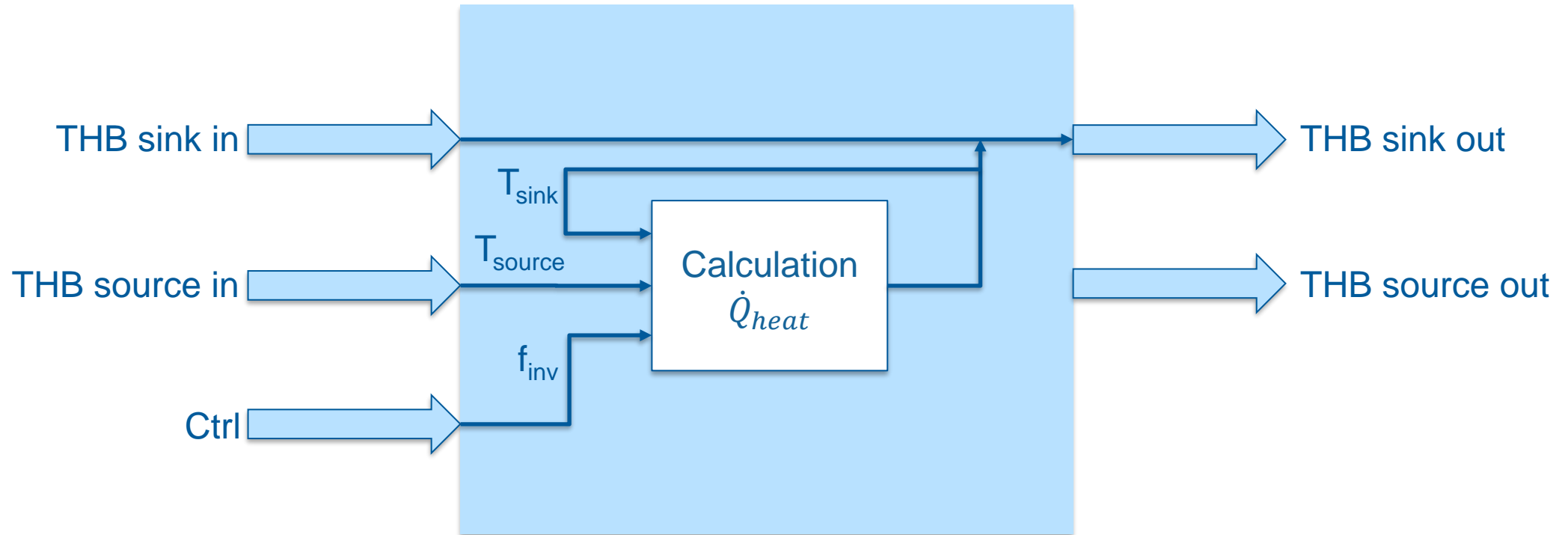
- **Black box model first: based on experimental data and fast**
 - Refrigerant cycle simulation might follow later for verification and inter- as well as extrapolation
- **Many degrees of freedom: 3 temperatures (ambient air, brine, heating system/DHW) + 2 frequencies $\rightarrow 3^5 =$ up to 243 measurements for quadratic interpolation**
- **Solution: Comparison of parallel operation with overlying single source operation at certain boundary conditions**
 - Correction factors required

$$\boxed{\text{ASO}} * \boxed{f_{\text{air}}} + \boxed{\text{GSO}} * \boxed{f_{\text{ground}}} = \boxed{\text{PO}}$$

2. Modelling approach



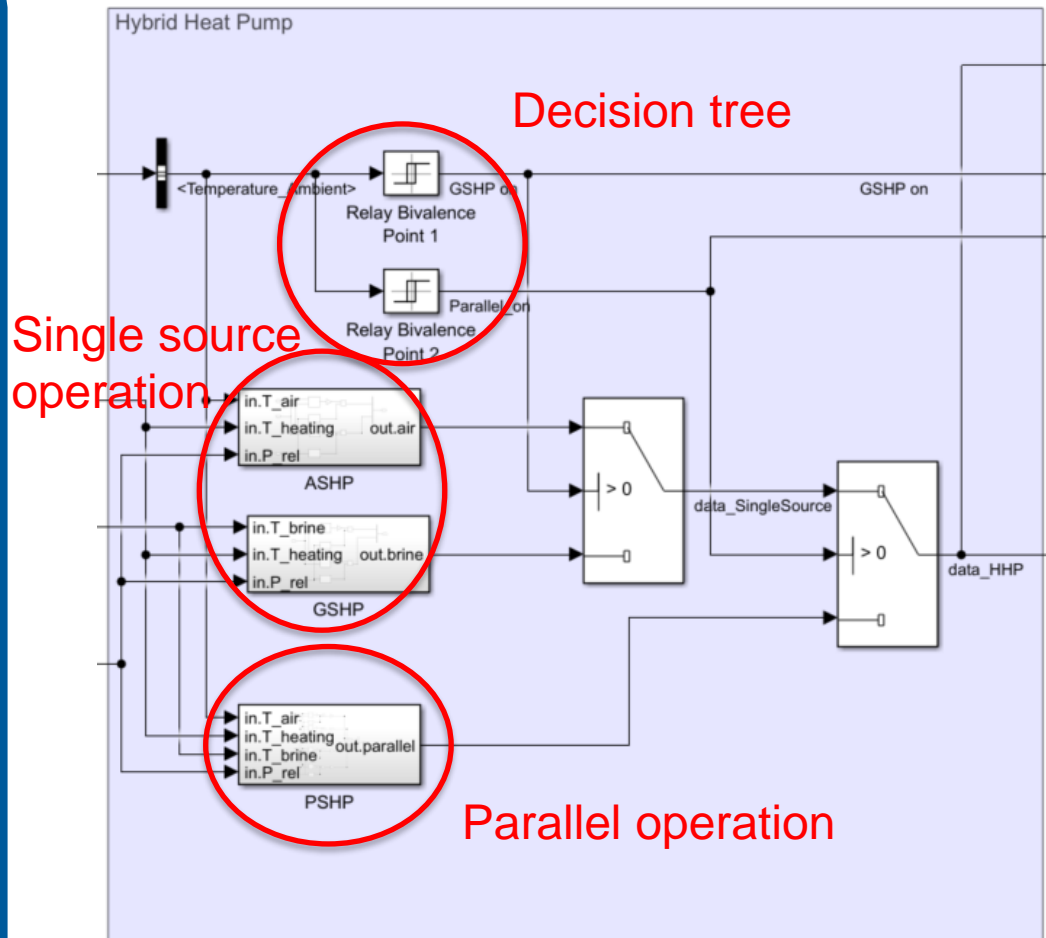
- First tries with CARNOT heat pump model were difficult, because of the internal feedback of the heating temperature T_{sink}



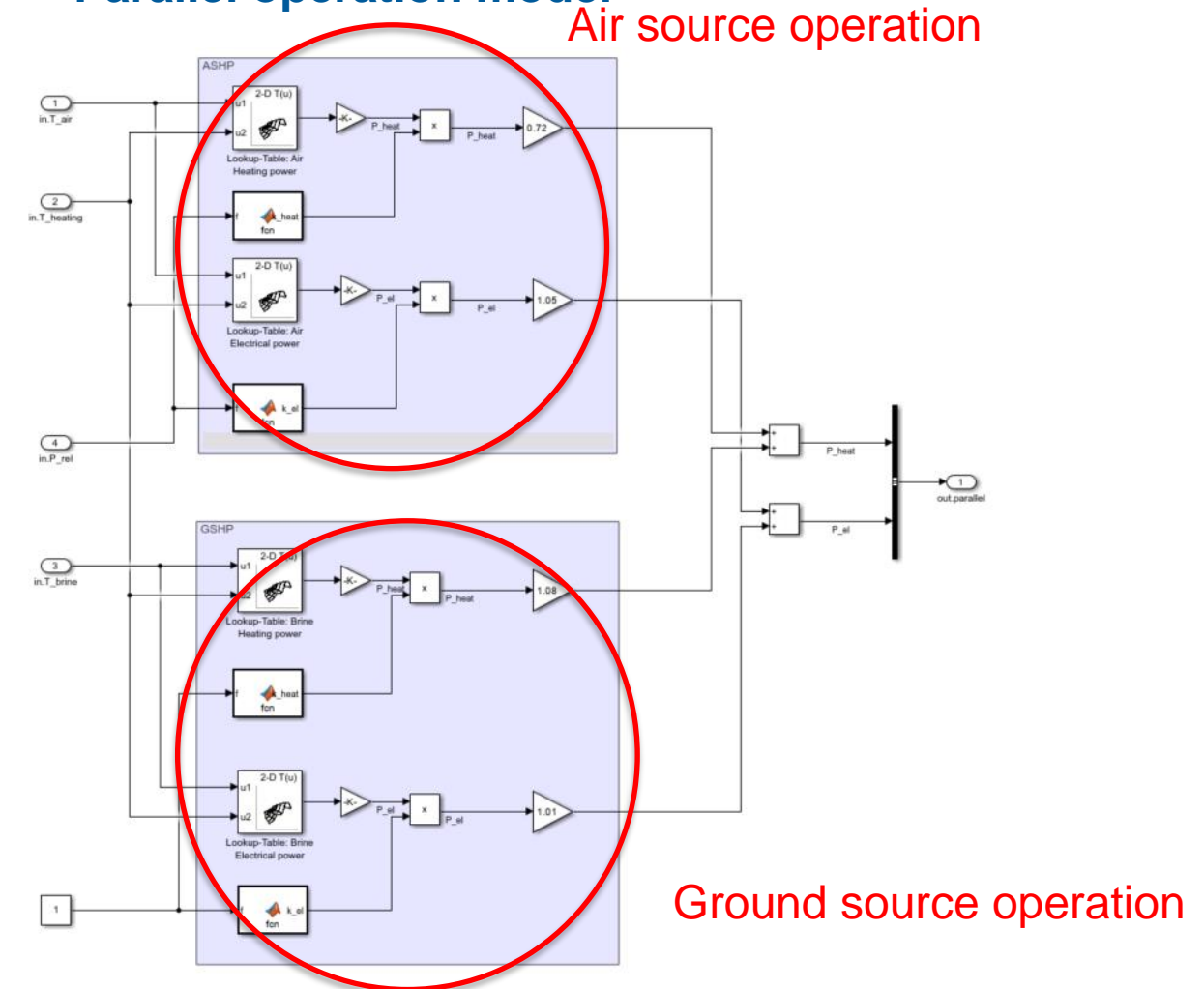
- But T_{sink} needs to be calculated with the sum of both heating powers! → own solution for now...

2. Modelling approach

Overall DSHP model

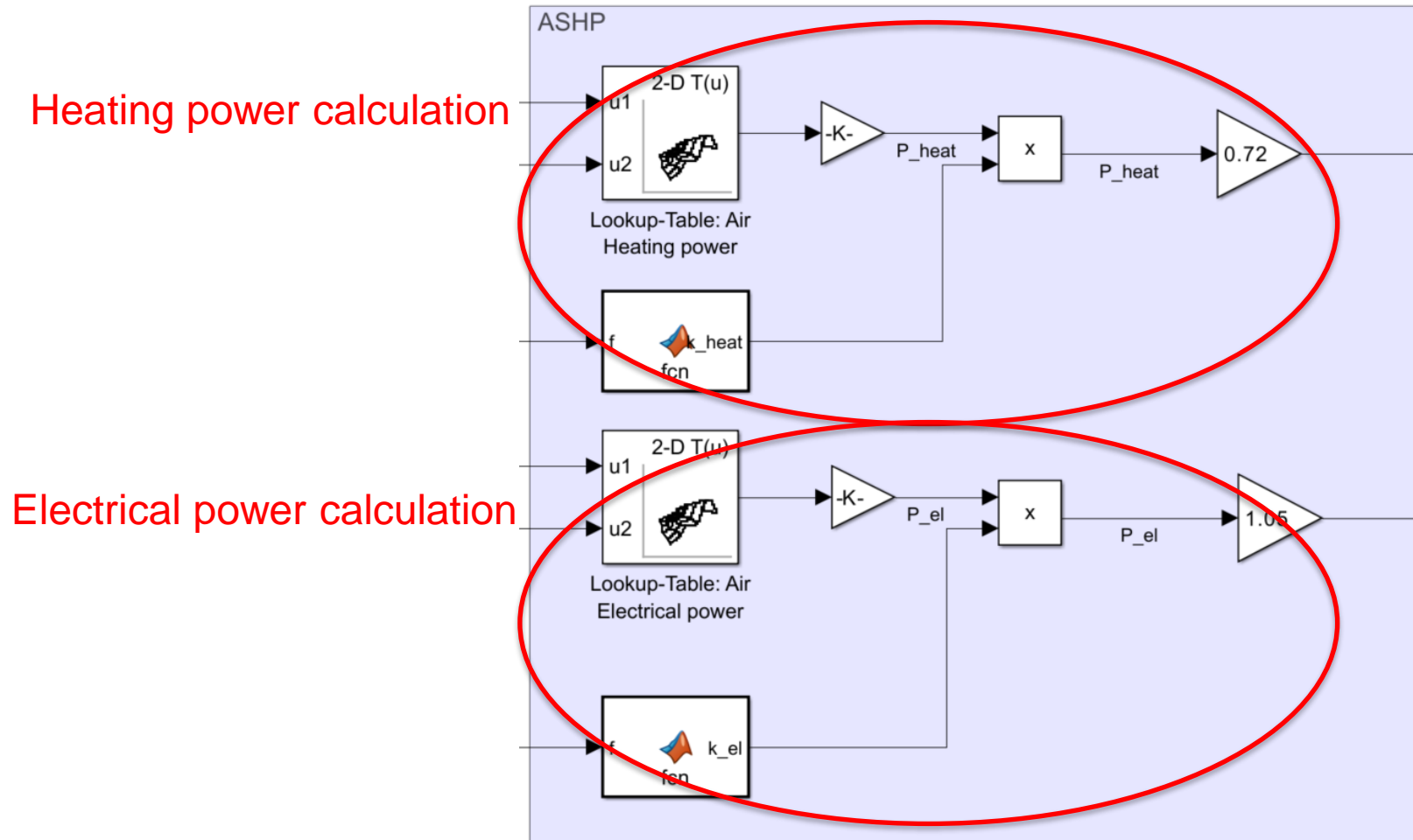


Parallel operation model



2. Modelling approach

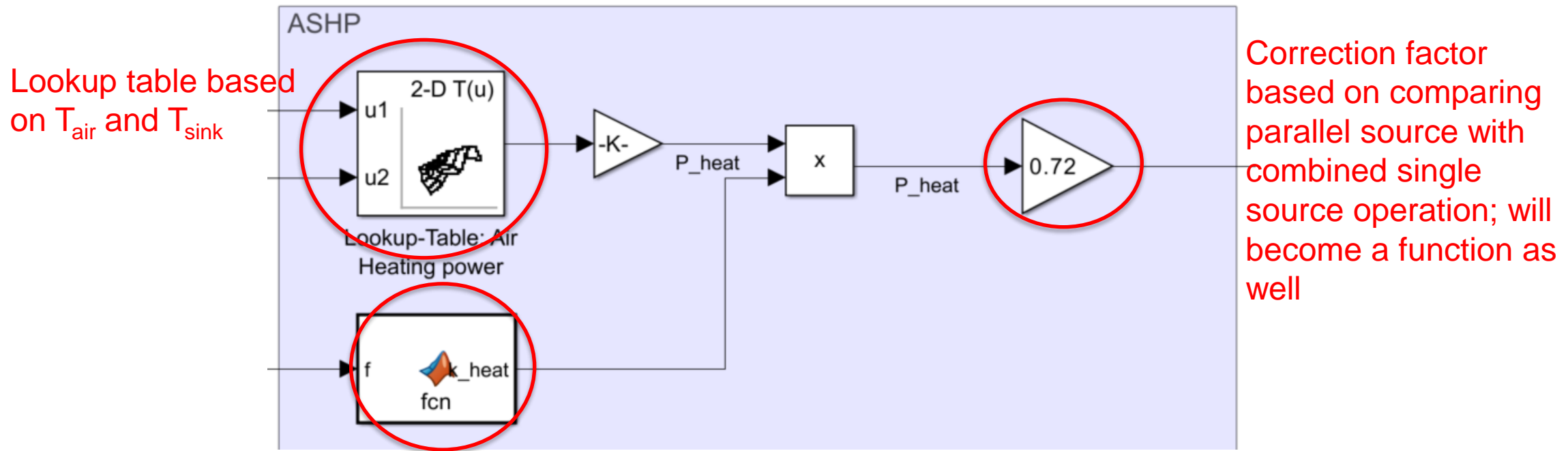
- Parallel operation: single air source operation plus correction factor f_{air}



2. Modelling approach

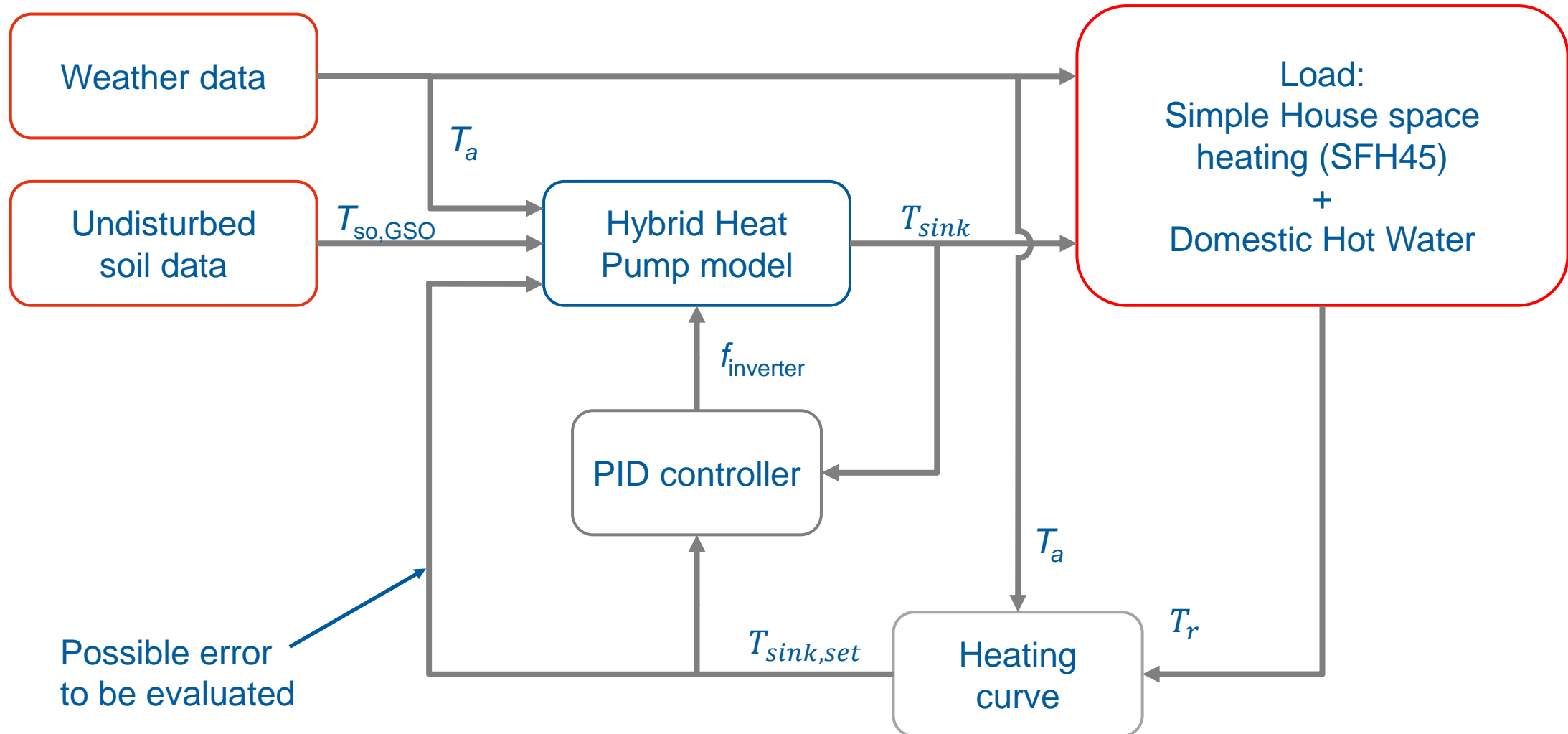


- Parallel operation: single air source operation plus correction factor f_{air}

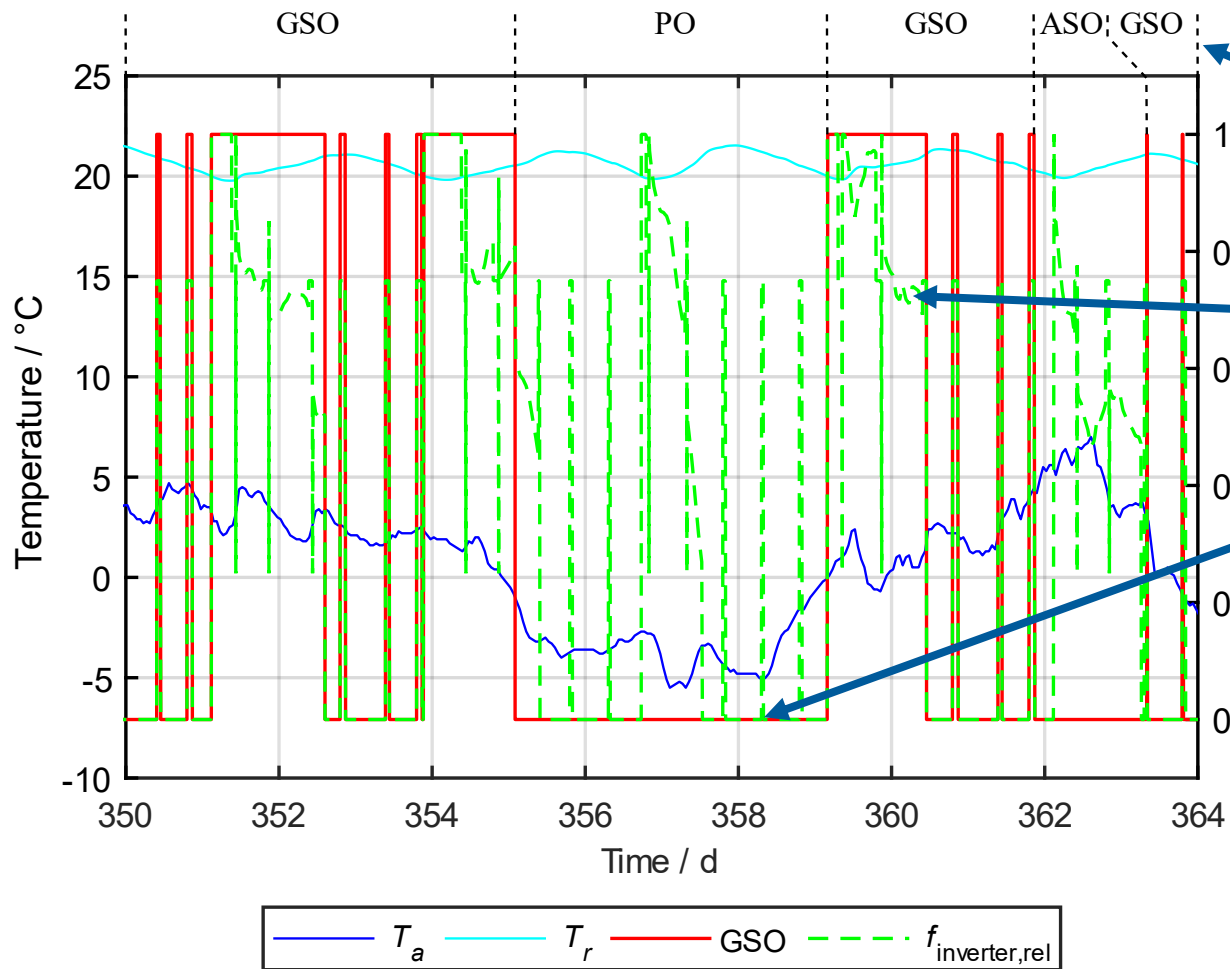


3. First results

Annual simulation



3. First results



1. Switching between the operation modes worked according to T_a
2. Inverter control operated quickly and stable
3. PO activated at too high T_a : inverter reduces air source operation to minimum and T_r still overheats
 → examine bivalence point 2 according to ground source temperature/ time of the year/ room temperature/ inverter frequencies...

Thank you for the attention!

Any remarks? Any questions?

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3. First results

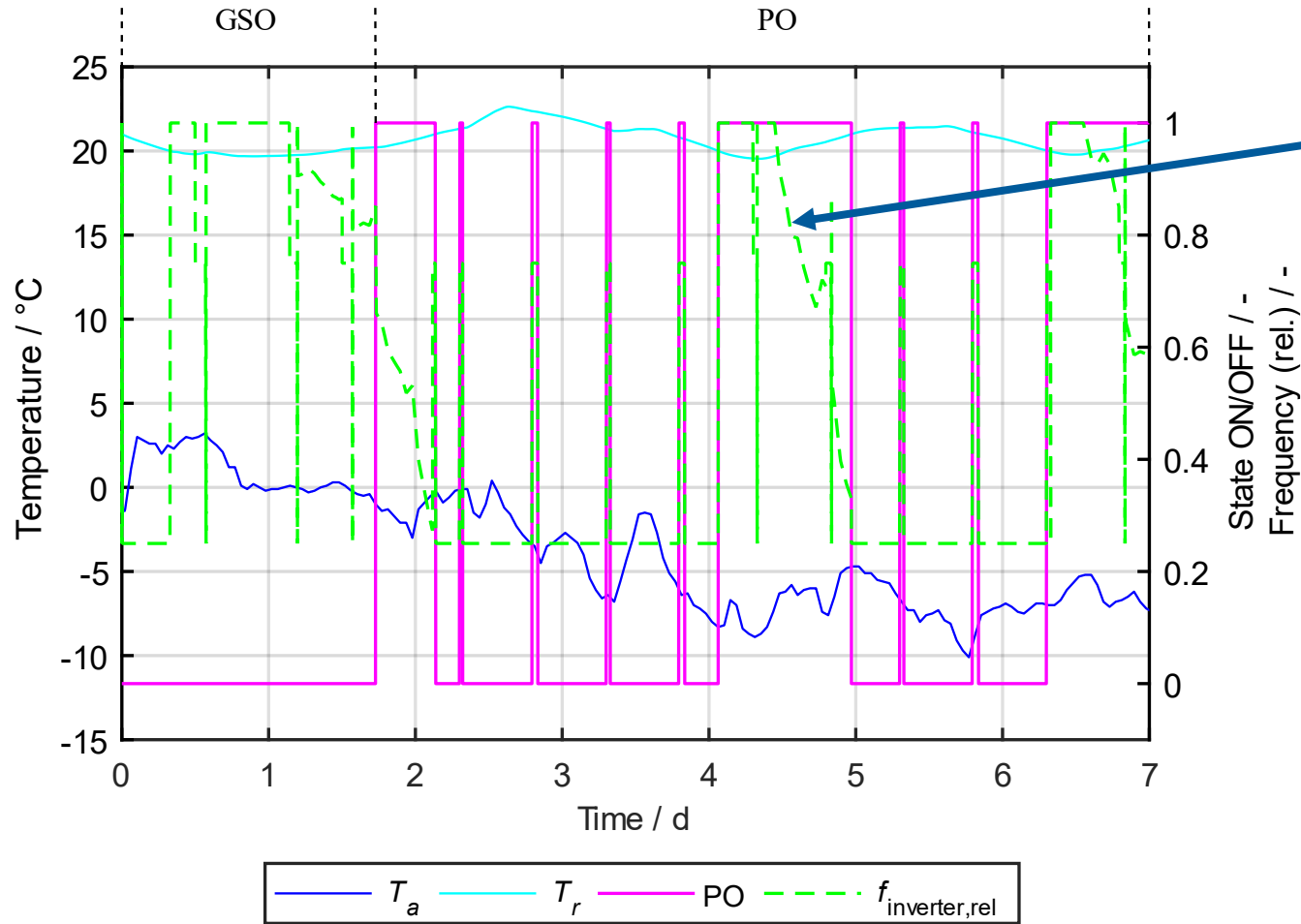
SFH45 + DHW according to DIN EN 13203

Value	Unit	DSHP	GSHP
Heating energy	MWh	12.8	
Seasonal Performance Factor	-	3.34	3.50
Energy extracted from the GSHX	MWh	4.42	8.70
Cooling power on GSHX	kW	5.53	6.37

- **SPF decreases slightly by about 5 %**
- **Energy extracted from GSHX reduced to about 51 %**
- **Cooling power on the GSHX reduced to about 87 %**



3. First results



4. PO activated properly at T_a : inverter controlled air source operation → examine bivalence point 2 according to ground source temperature/ time of the year/ inverter frequencies



3. PhD project

First experiments – parallel operation

At ambient temperatures of -10°C :

$$\text{COP}_{\text{GSHP}} > \text{COP}_{\text{DSHP}} > \text{COP}_{\text{ASHP}}$$

But cooling power of the ground source:

$$P_{\text{DSHP}} < P_{\text{GSHP}}, \text{ to about } 65\%$$

